







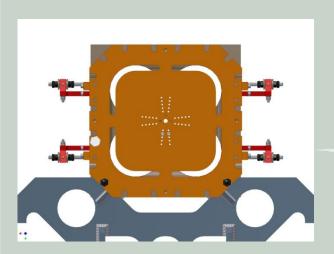




RFQ Assembly Plan #3

August 2011

by P. Savage













Front End Test Stand (FETS) RFQ assembly sequence

We need a step by step plan to follow to enable us to successfully assemble the FETS RFQ so that:

- 1. The vanes are aligned to the best of our ability.
- 2. We achieve a vacuum seal.

The assembly sequence can be broadly split into 4 parts:

- •Part 1: Vane to vane alignment without O ring in place.
- Part 2: Back to manufacturers for final machining.
- Part 3: Section assembly with O ring in place.
- Part 4: Alignment of completed assembly w.r.t. the FETS.

Note that I have not yet considered where the bead-pull tests come in this sequence.











Part 1

Vane to vane alignment without O ring in place.

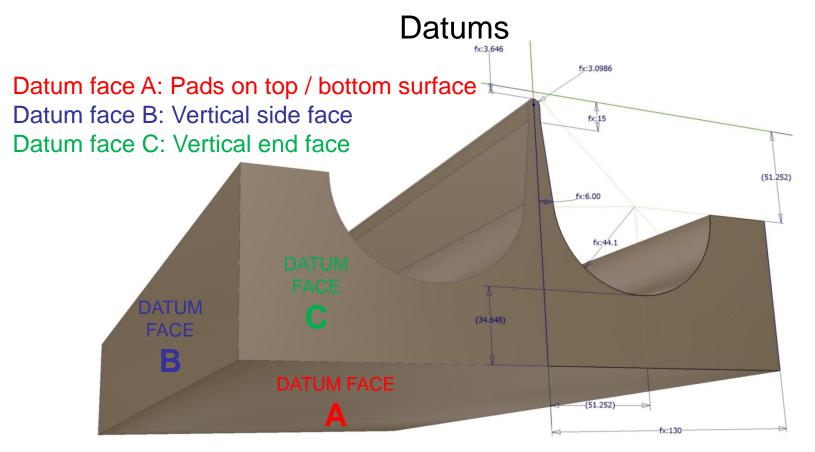














- •during manufacture of individual parts.
- during assembly of each one metre section.
- •for section to section installation on the FETS.





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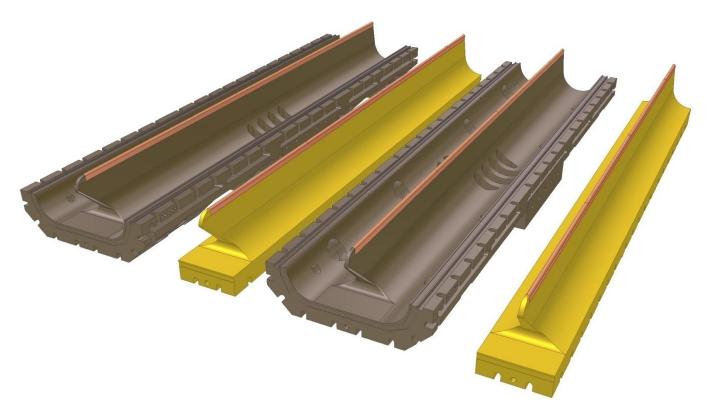








Assume 2 major and 2 minor vanes are manufactured



- •2 major vanes and to minor vanes for RFQ section 1 arrive at RAL.
- Moved into R8
- •They have been manufactured to tolerance.
- •They are complete except are all 2mm over-length, 1mm at each end.



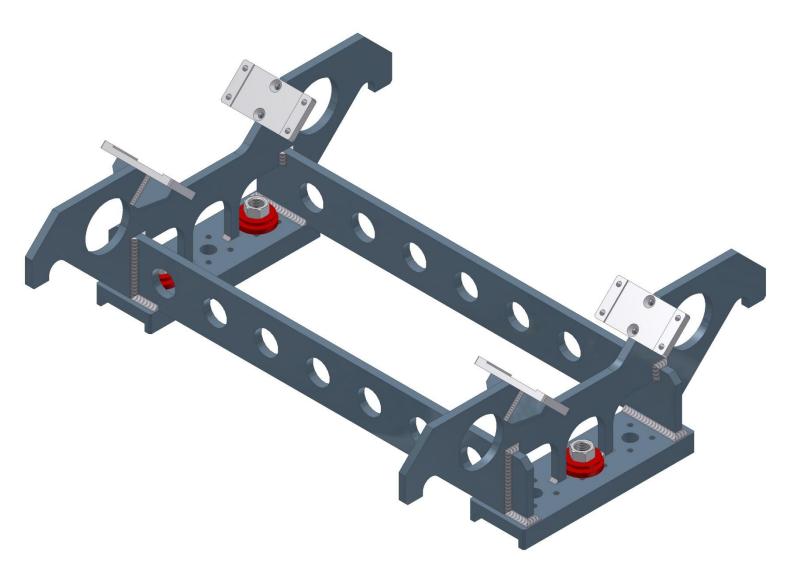








RFQ section will be assembled into the cradle





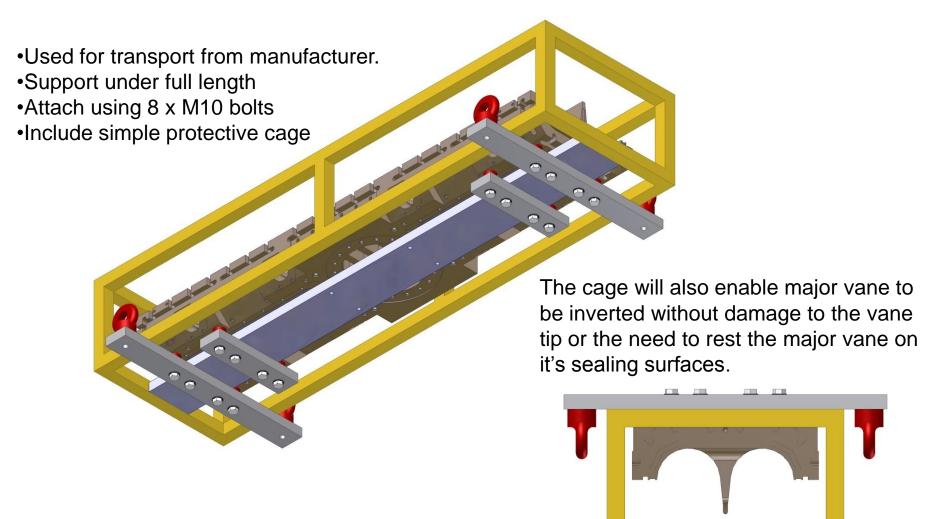








Concept design for Major Vane lifting beam.





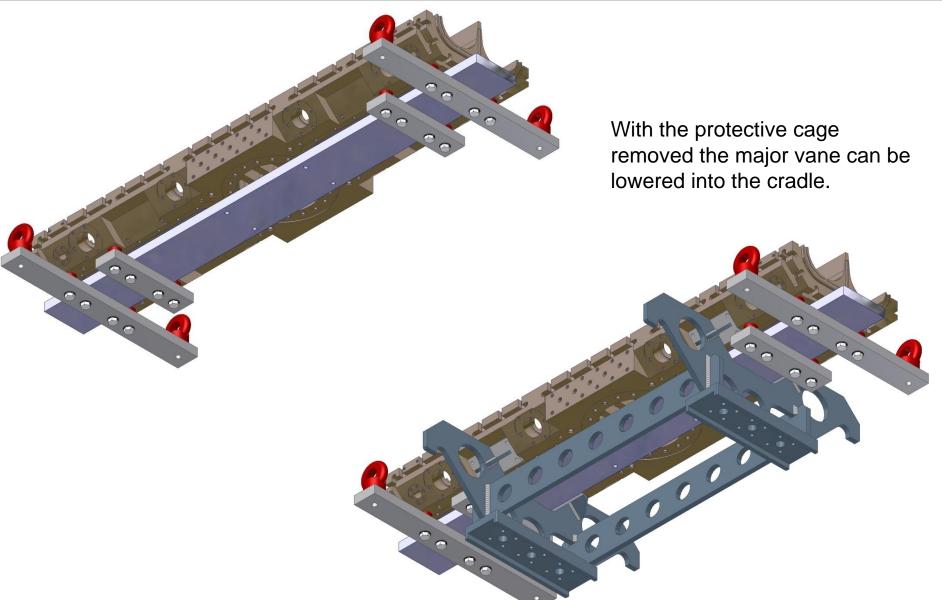














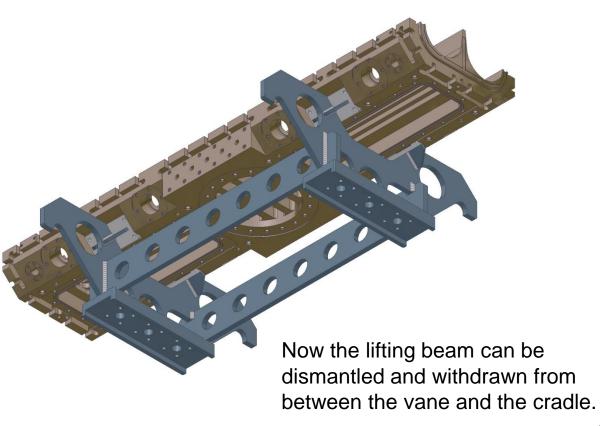




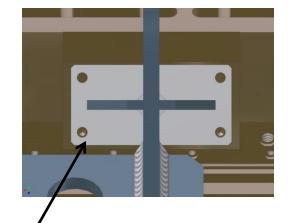




Lifting beam unbolted and removed



The vane can now be bolted to the cradle.





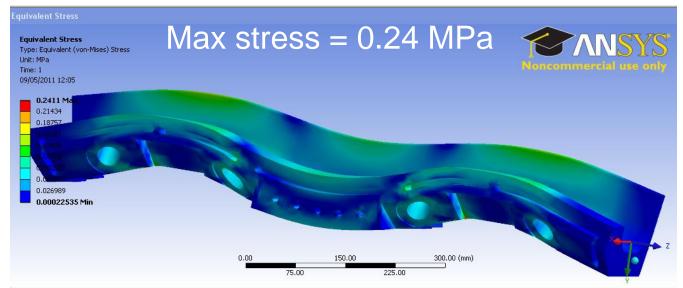


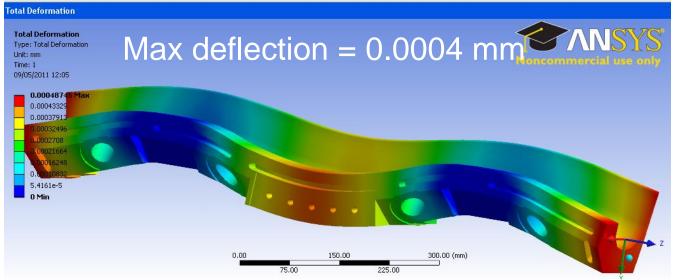






Will the major vane sag when resting in it's cradle?



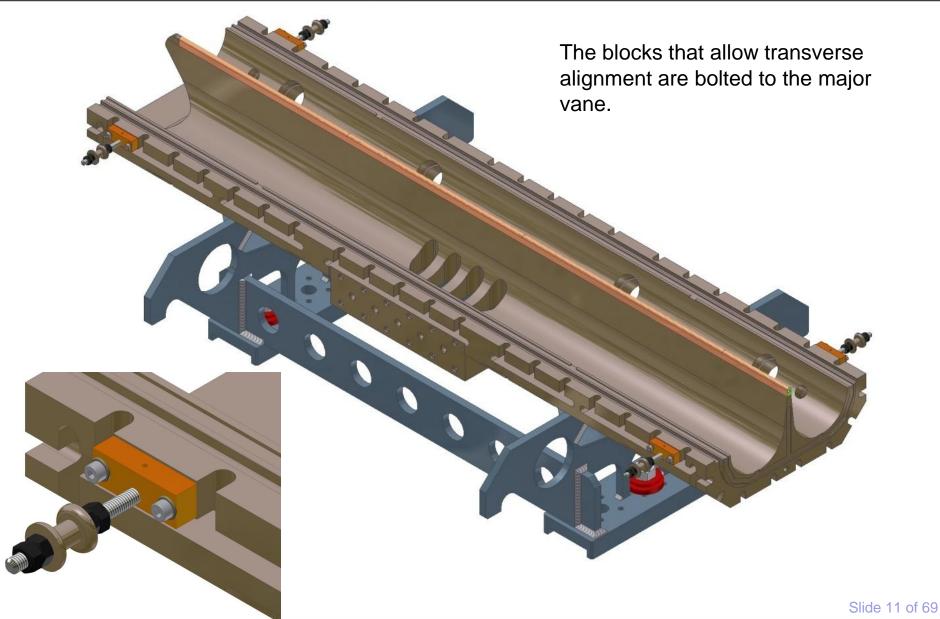














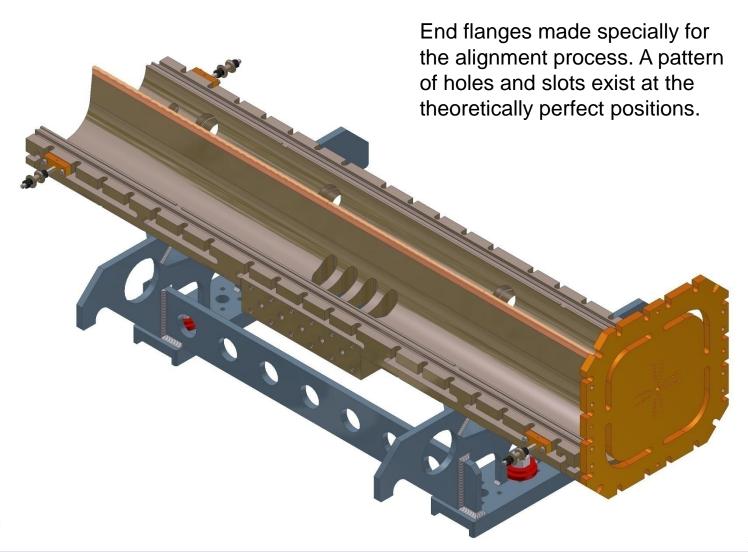








Add alignment end flanges



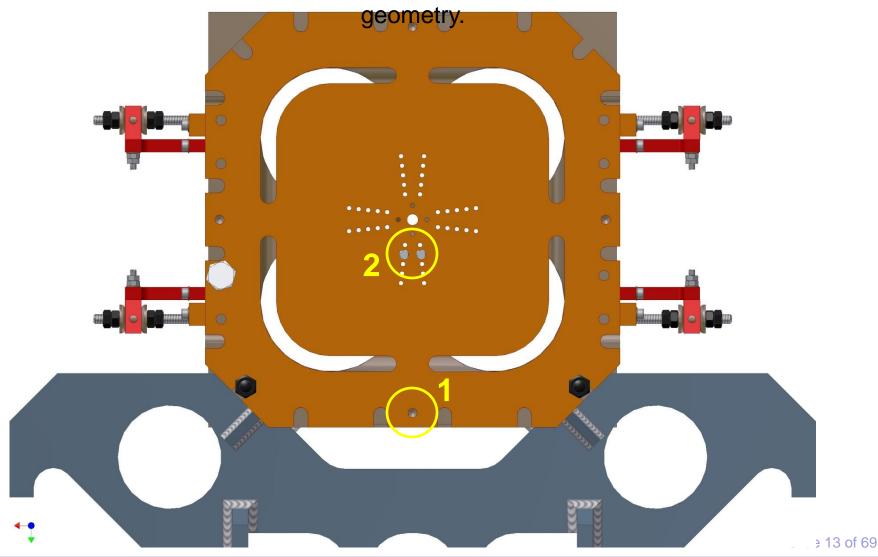








End flange is first doweled to the major vane (1) before being aligned to the vane sides using tapered dowels (2) - close to the beam transport region, in an area of unchanging





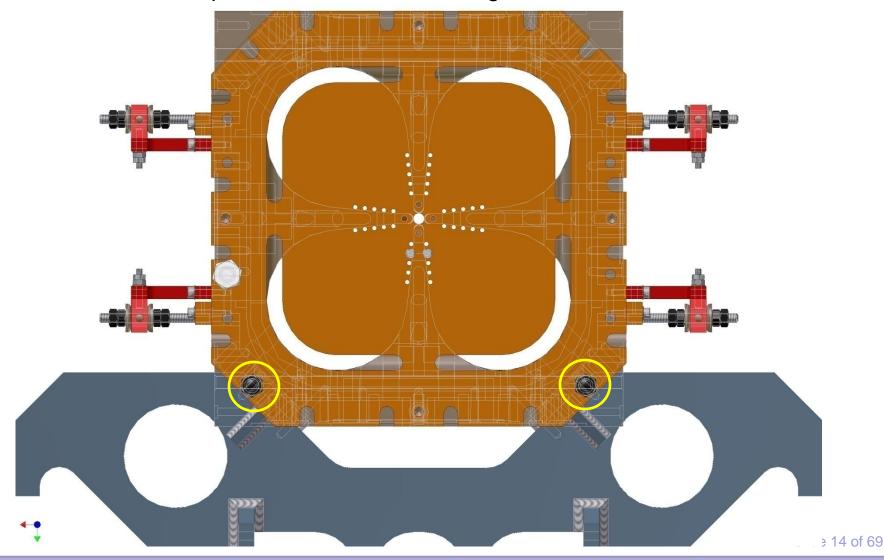








Once the end flange is constrained it can be bolted to the major vane – checking all the time that the tapered dowels are not being forced into the vane sides.





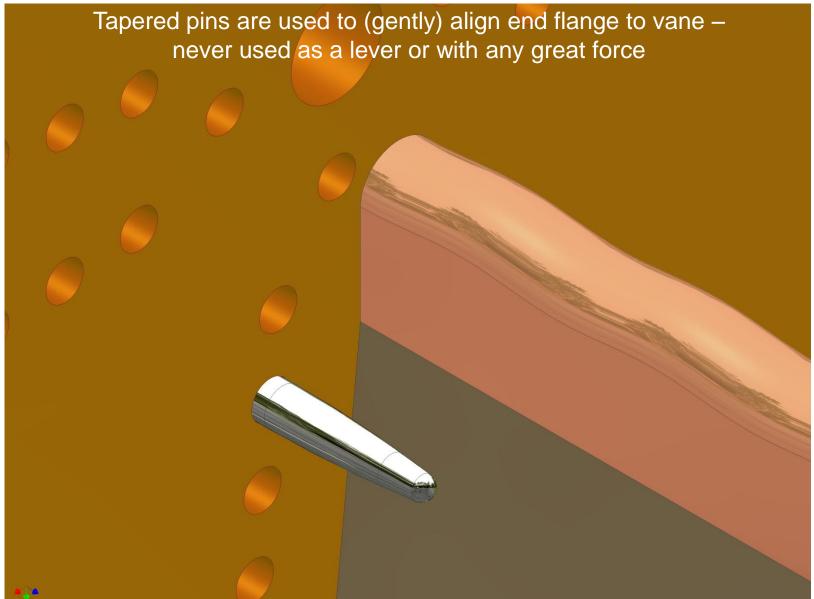
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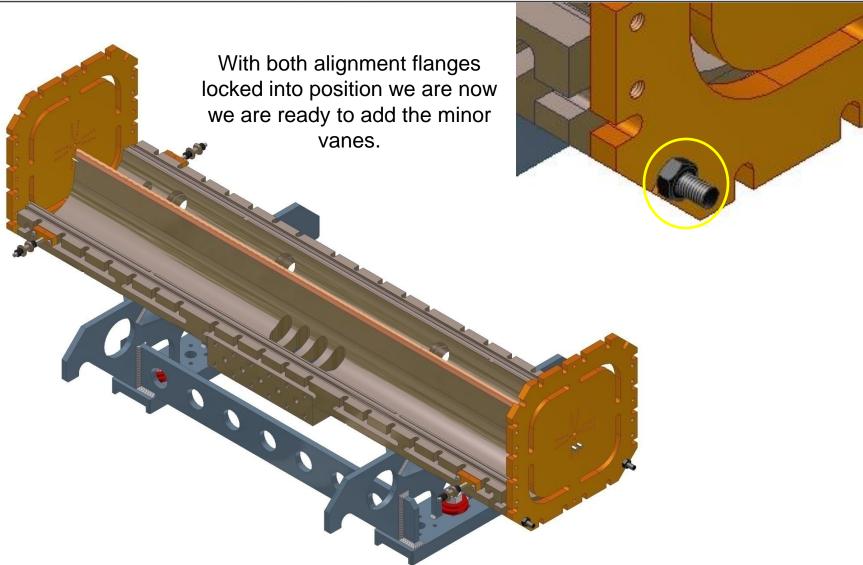












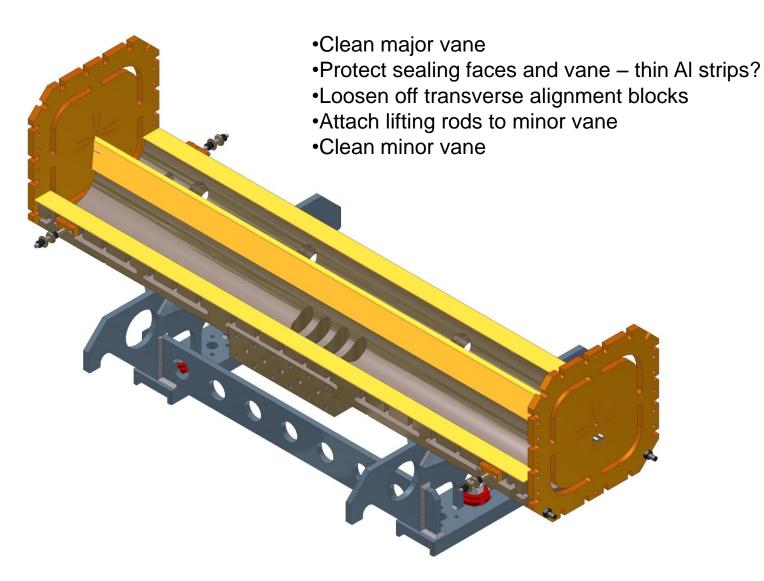












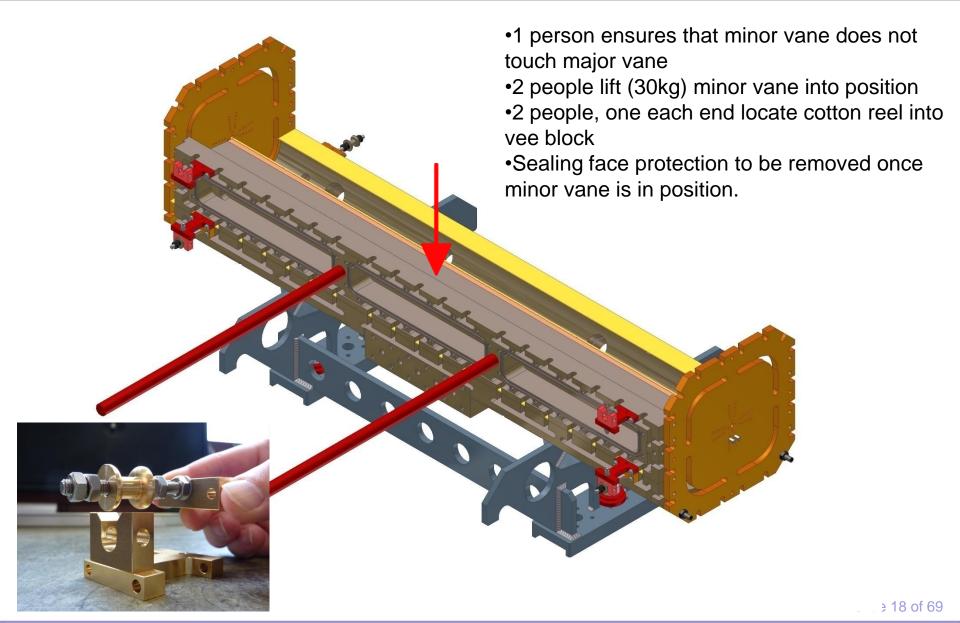














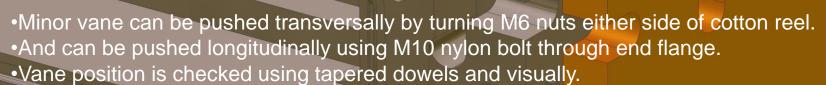






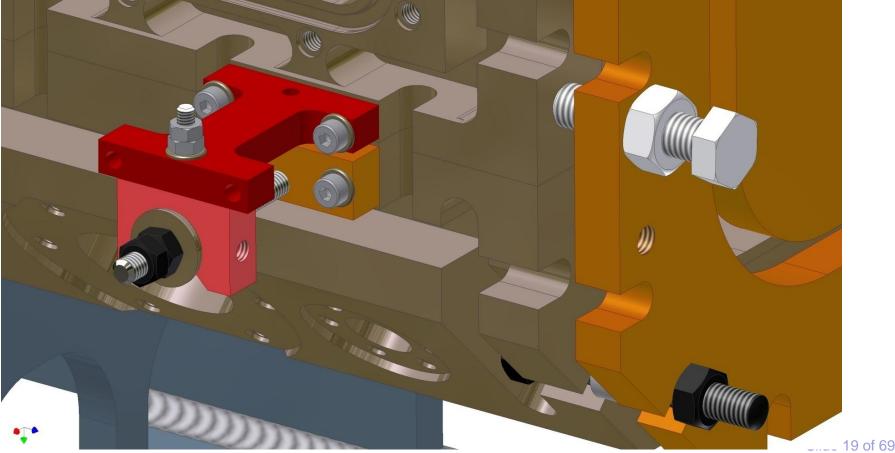
Imperial College London





Once in position adjustment pieces are locked into position

•(Using shim to allow fixed clearance and hence vane removal and replacement)









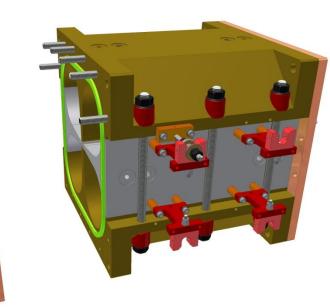




Transverse positioning blocks

Concept of transverse positioning blocks is about to be trialled on the RFQ scale model.







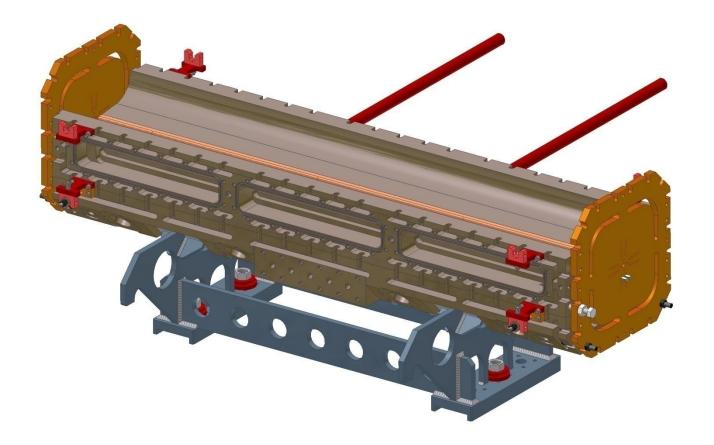








Repeat procedure for second minor vane







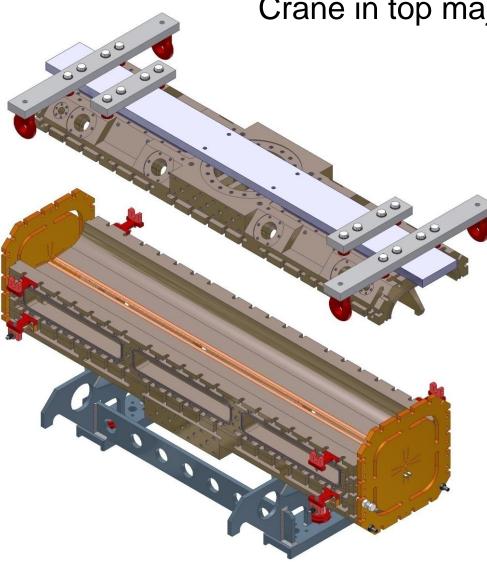
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- •Clean
- Protection for vanes and sealing faces required.
- Use chain block for sensitivity NOT the R8 crane
- One person at each corner
- •Rest on protective strips before finally lowering into position.

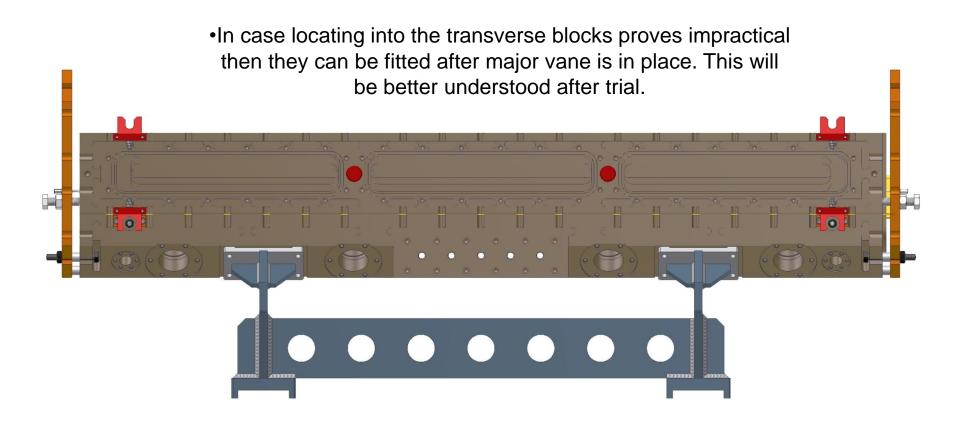












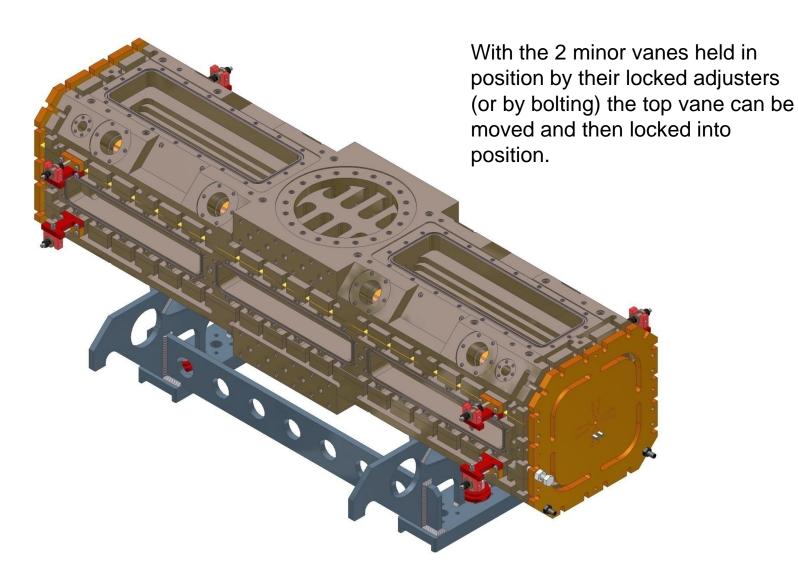












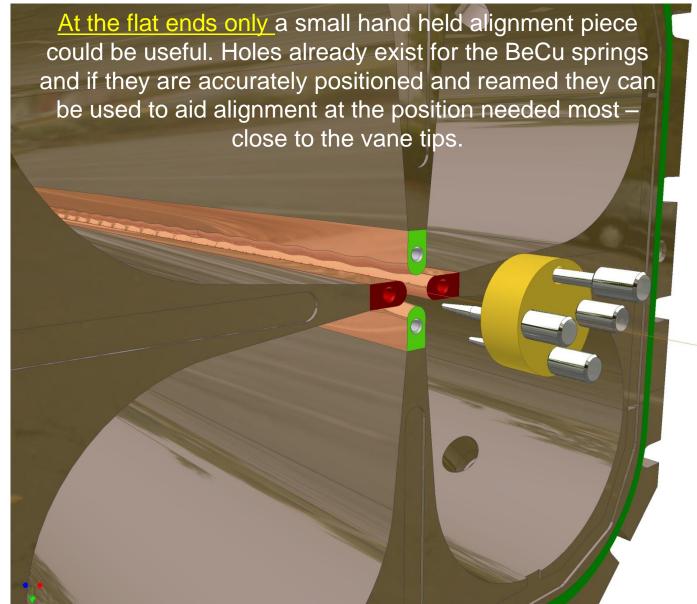












RF test required.

Need network analyser and loops for input and output.

If dipole modes are close will need to find out which vanes to move by performing bead pull test.

Bead pull test – simple (Imperial) one may suffice? If vanes must be moved then unbolt and adjust. Repeat bead pull test.

Blanked off tuner ports will be sufficient. Ideally with tuner plugs in place.



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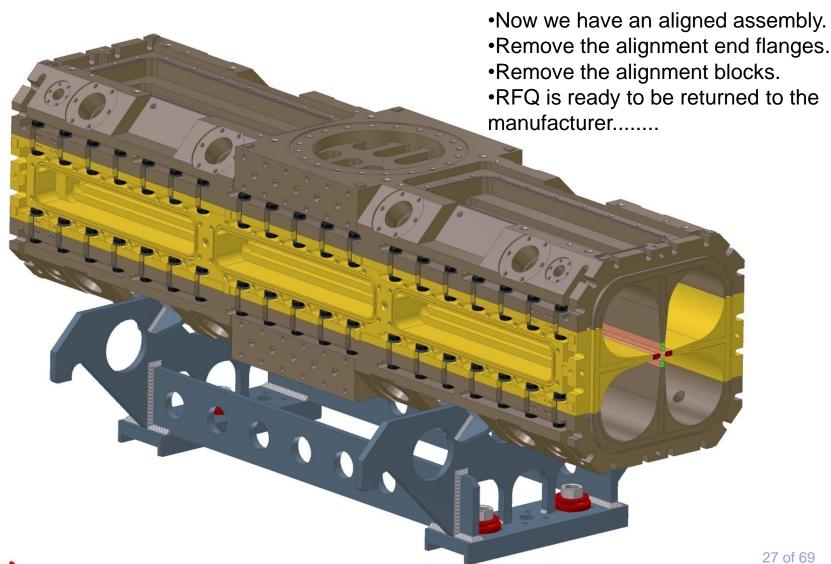








Ready to be returned to manufacturer



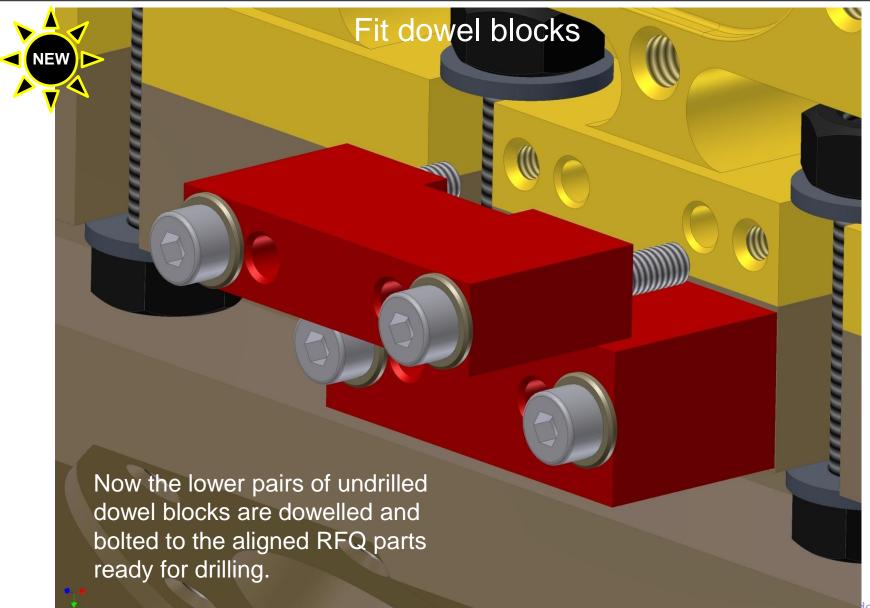






















Part 2

Back to the manufacturers for final machining.



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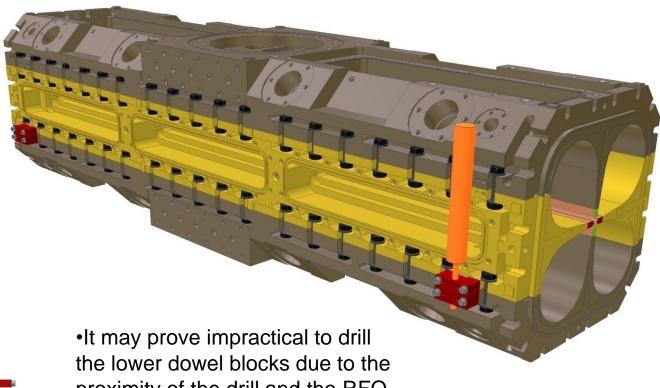


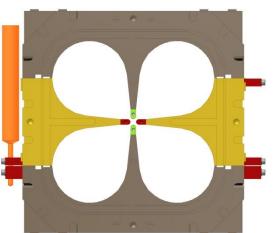






Drilling lower dowel blocks





proximity of the drill and the RFQ body. If so they can be extended and supported underneath and drilled further away from the RFQ

body.

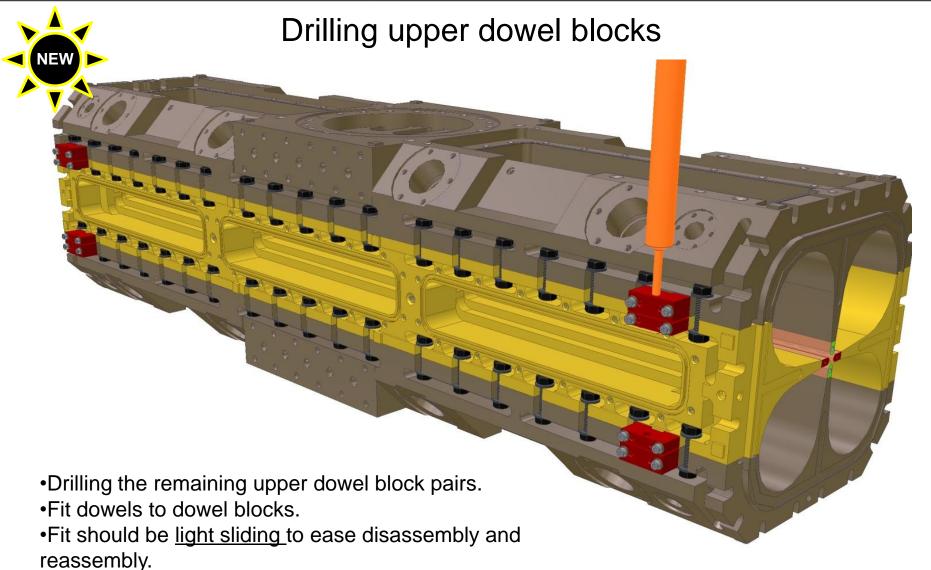














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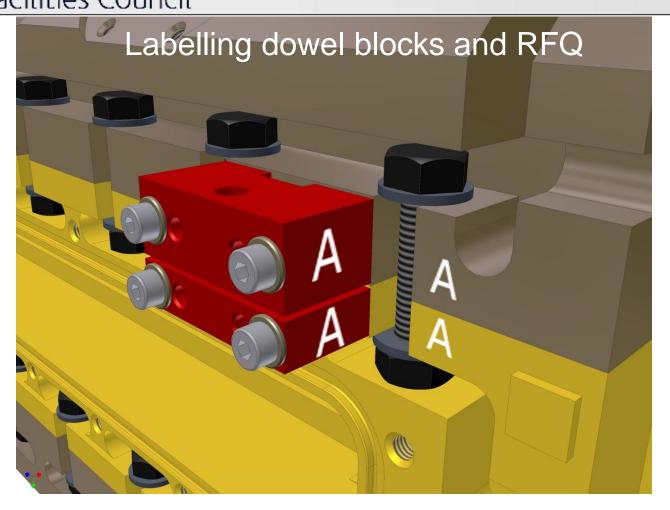












Each dowel block pair now has a unique <u>orientation</u> and <u>position</u> on the RFQ assembly. They may never need to be removed but in case they need to be replaced by the alignment blocks they must be labelled.



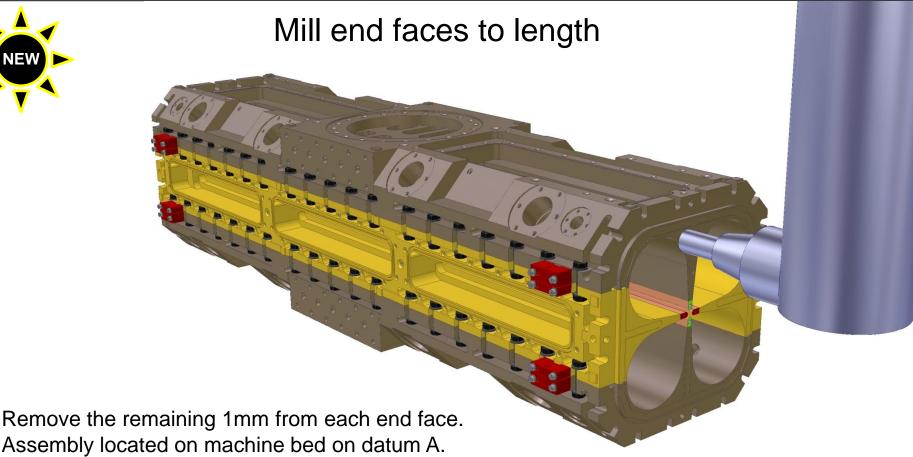












End faces are:

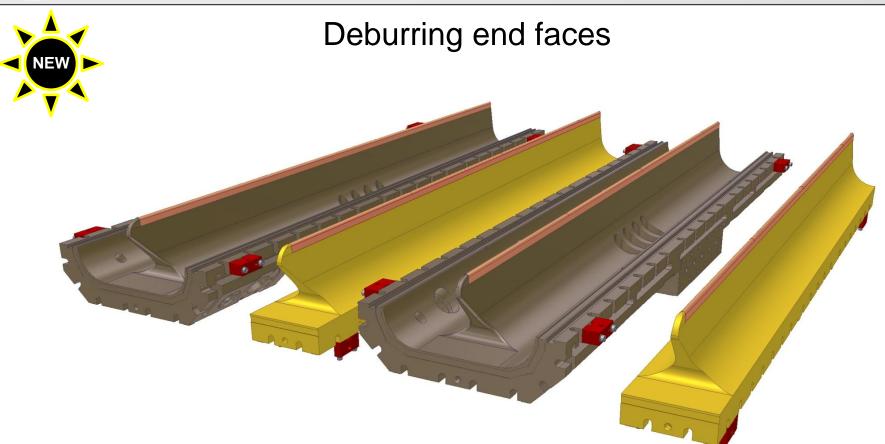
- Perpendicular to datum face A
- Parallel to each other
- To the final length
- •Can final operation be done without coolant to minimize contamination?











Now the finished pieces of RFQ section 1 can be dismantled leaving the drilled & labelled dowel blocks in position. The new machining of the end faces must now be deburred. This stage can be done either at the manufacturer or at RAL. We are now ready for reassembly with the O ring in place.











Part 3

Section assembly with O ring in place.



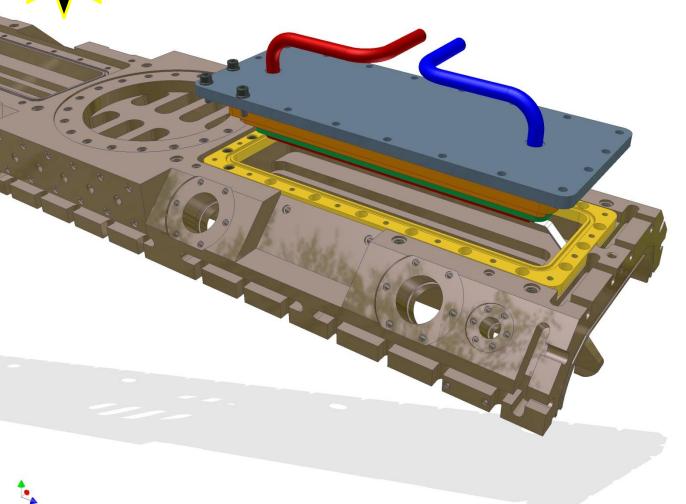












It will be easier to fit the baffles into the lower major vane with it inverted.







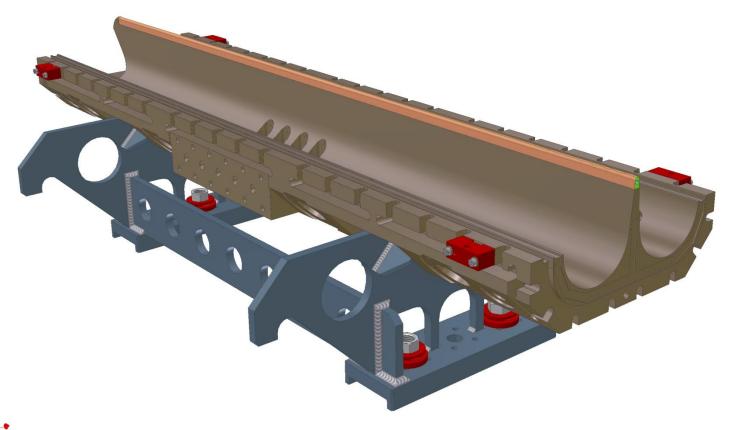






Start assembling RFQ section 1

- Lower major vane into cradle
- Clean major vane







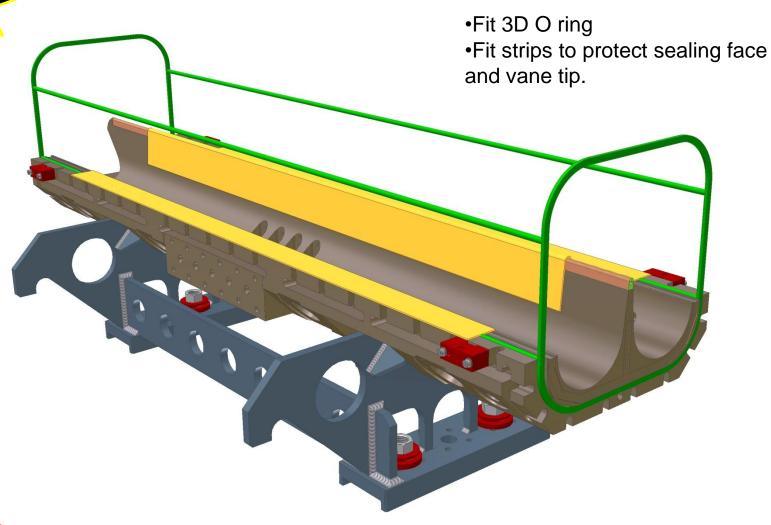






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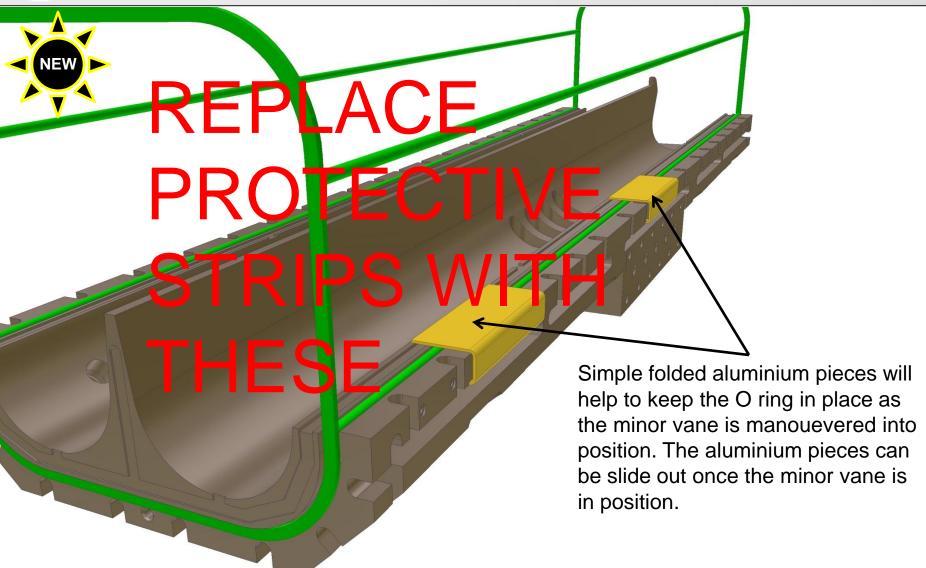










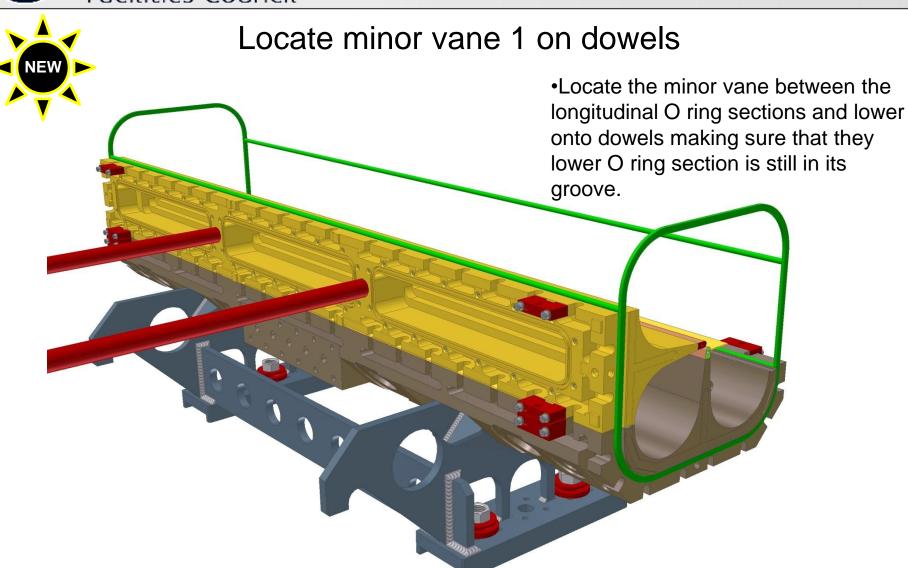
















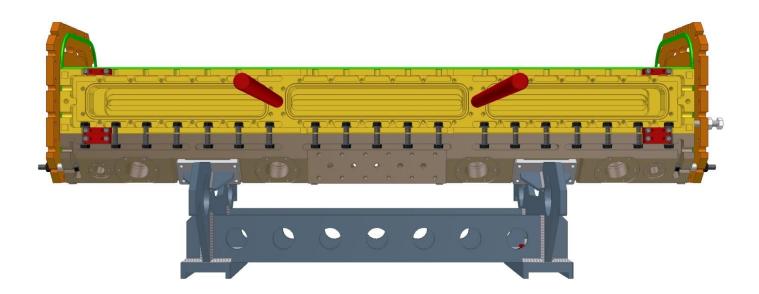








Start to bolt down minor vane 1



•Fit the end alignment flanges – they are there this time just to prevent the O ring from squeezing out during compression as the minor vane is bolted down.





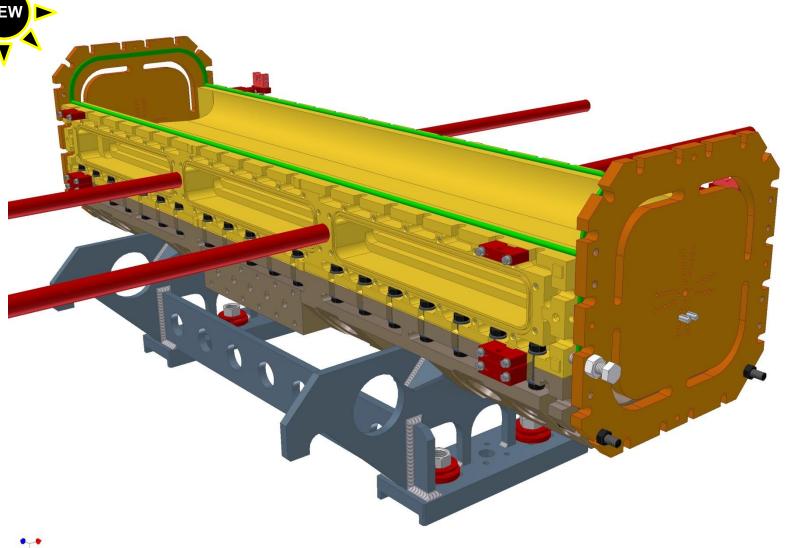














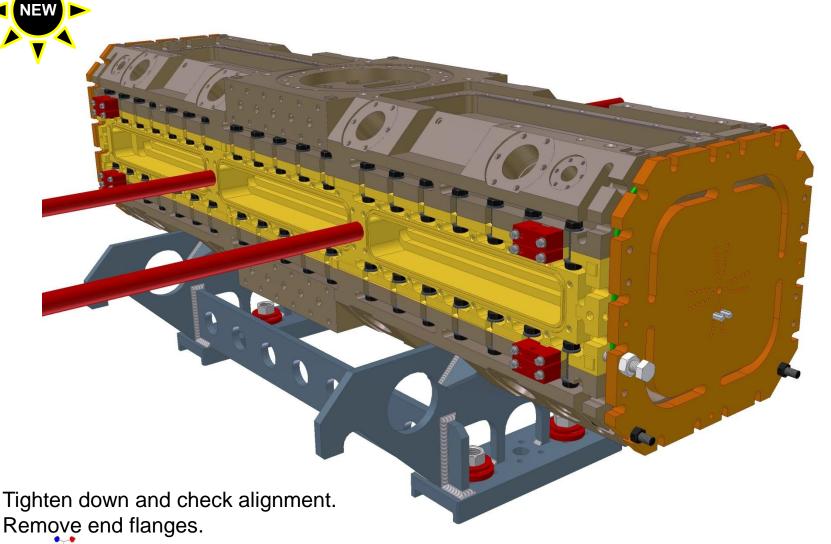












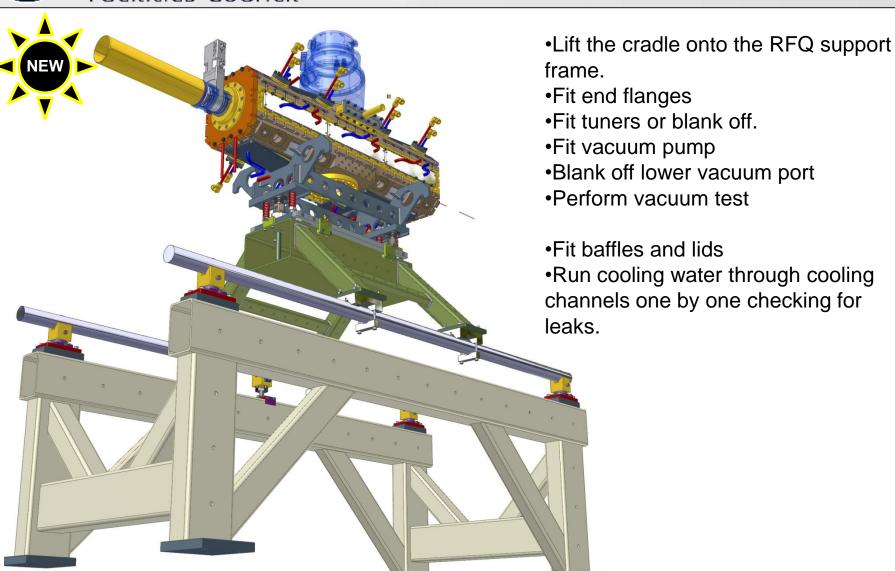














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Load due to atmospheric pressure and weight of vacuum pump

- Quarter de-featured model
- •3 symmetry planes
- •Nodes: 24,500

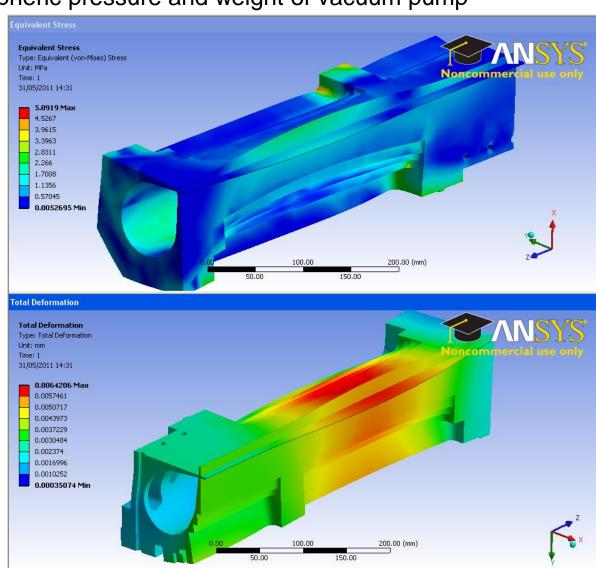
Loads:

- •100,000 Pa atmosphere
- •300N vacuum pump

Results:

Max. Equ stress: 4.6MPa In region of cooling pocket internal corners.

Max. Deformation: 0.0065mm Vane tips move towards beam axis













Part 3

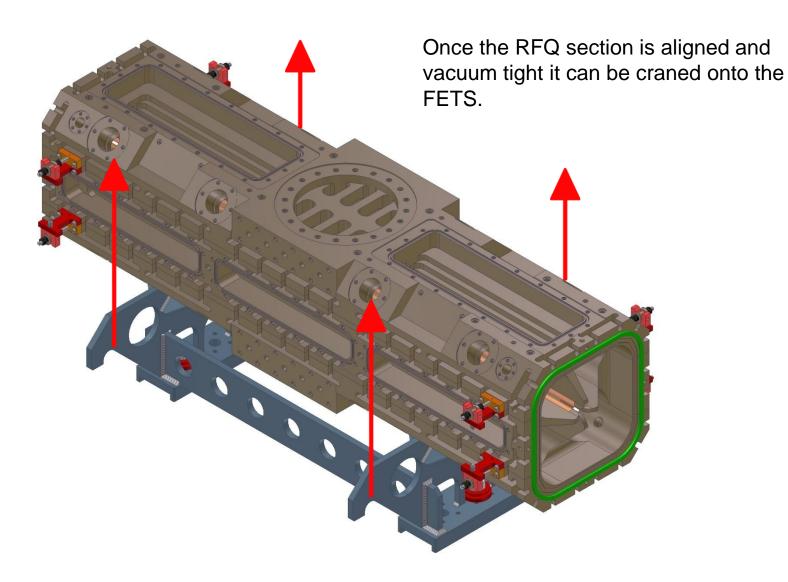
Alignment of completed assembly w.r.t. the FETS.













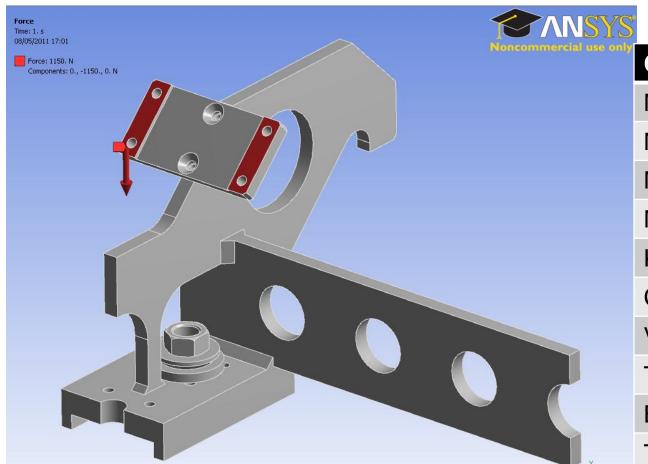








Structural steel, quarter cradle model, 2 symmetry planes, load = 1150 N



QN: What is the stress and deformation in the RFQ support cradle due to the weight of the RFQ and the ancillary components?

<u></u>	
Component	Mass (kg)
Major vane	80
Major vane	80
Minor vane	30
Minor vane	30
Kinematics	10
Cradle	20
Vac pump	30
Tuners x 16	80
Baffles	100
TOTAL	460
Load per 1/4	115



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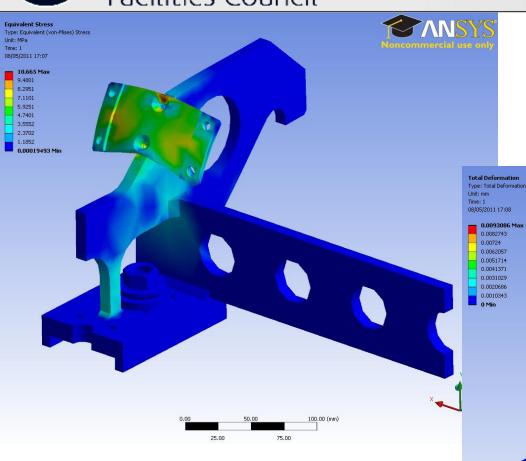




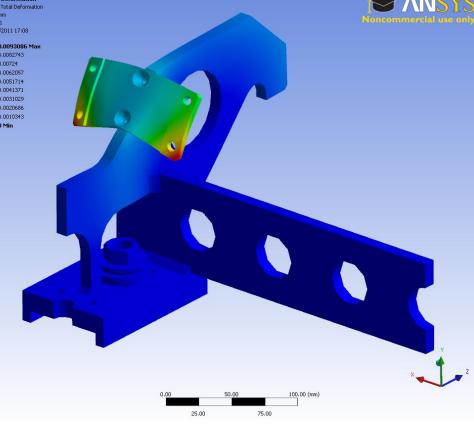








Maximum equivalent stress = 10MPa Maximum deformation = 0.01 mm





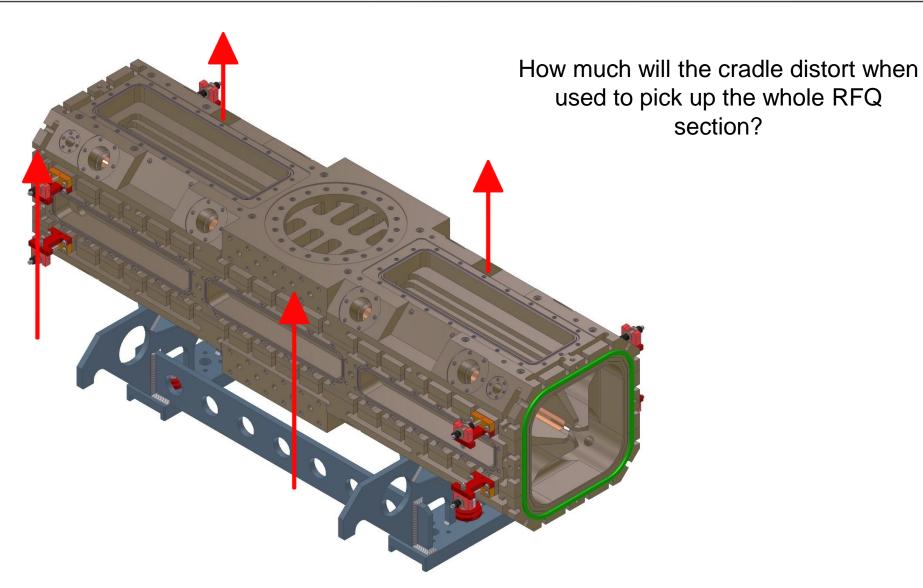




section?













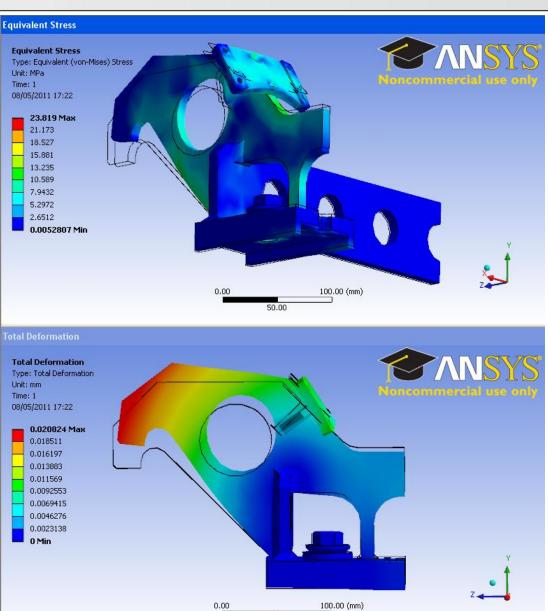




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50.00

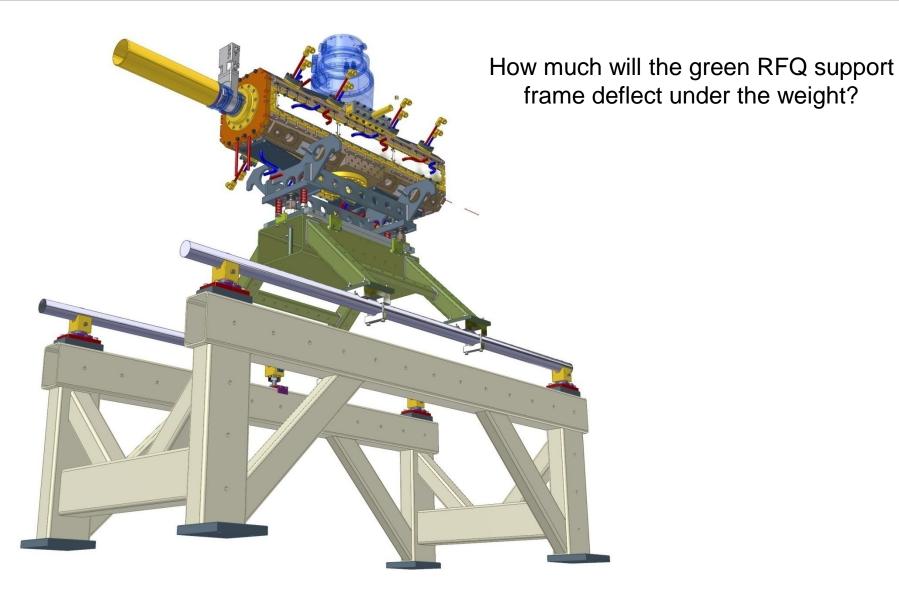














Time: 1. s 08/05/2011 16:30



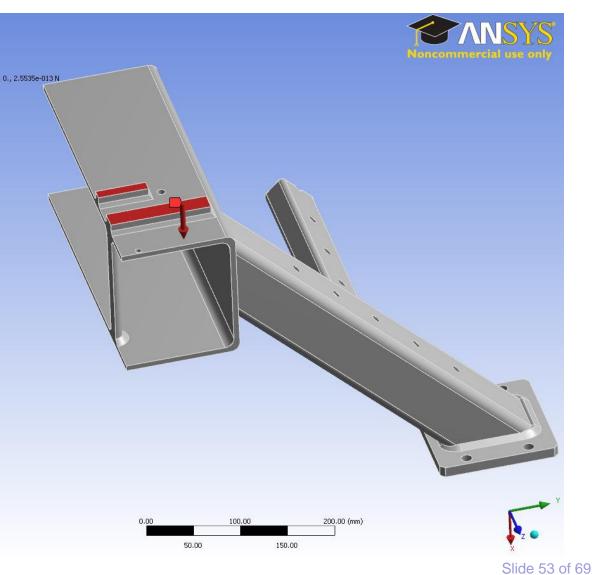






Structural steel, quarter cradle model, 2 symmetry planes, load = 1150 N

	Force: 1150. N Components: 1150.,
Component	Mass (kg)
Major vane	80
Major vane	80
Minor vane	30
Minor vane	30
Kinematics	10
Cradle	20
Vac pump	30
Tuners x 16	80
Baffles	100
TOTAL	460
Load per 1/4	115





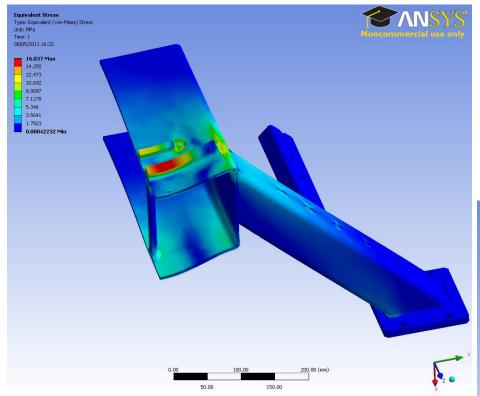
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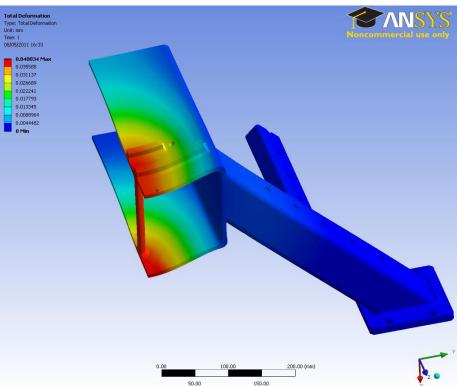








Maximum equivalent stress = 16MPa Maximum deformation = 0.04 mm



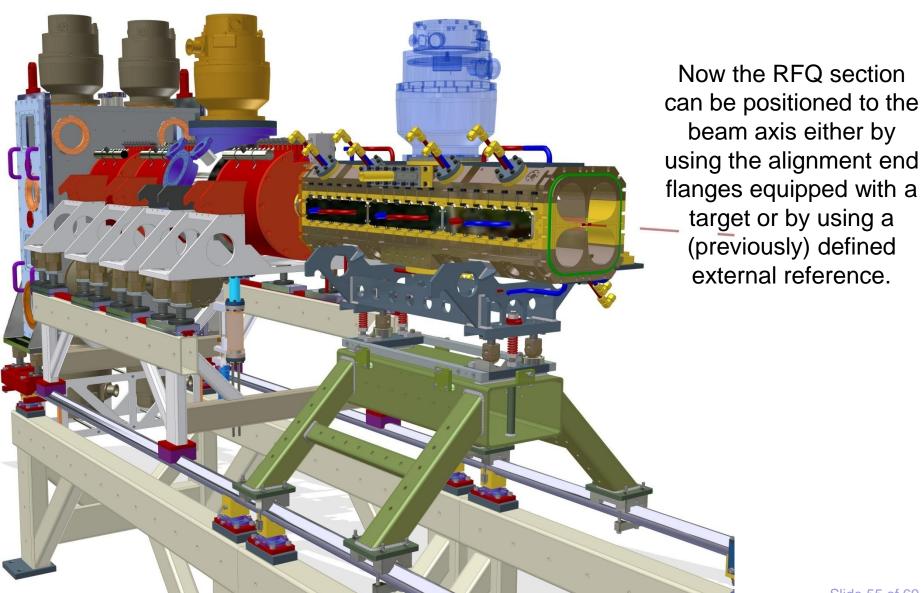




















Hypothetical Case 1

Supporting one major vane by the vacuum port flange.

We would never do this – it's just to get a feel for the stiffness of the vanes.



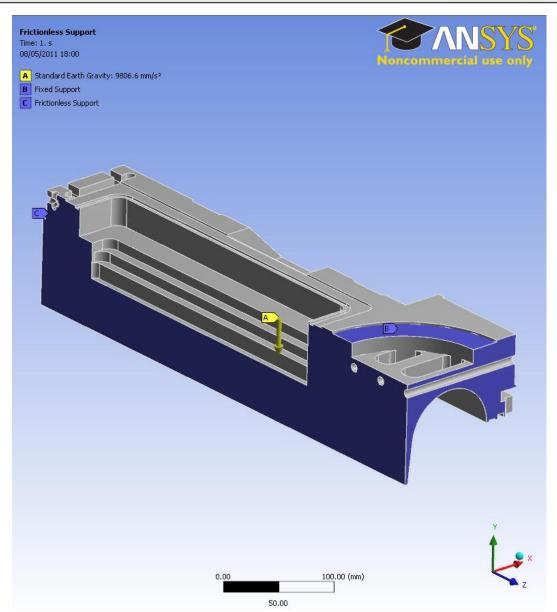
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The simulation model

Imagine we supported one RFQ major vane by the vacuum port flange.

QN: How much does the major vane sag under it's own weight?



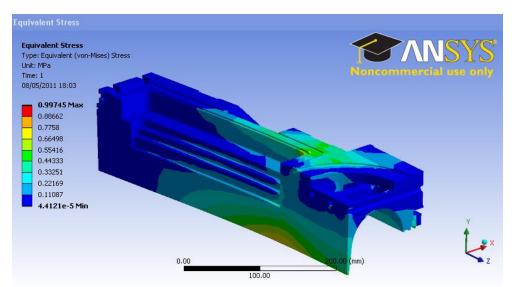


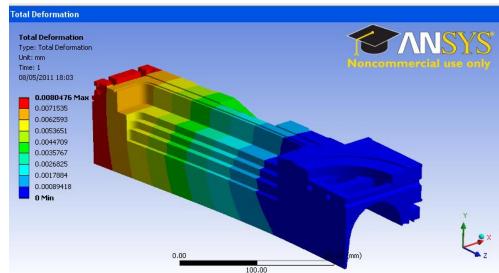




Max stress = 1 MPa

Max deformation = 0.008mm















Hypothetical Case 2

Supporting one major vane by the end (cantilevered).

We would never do this – it's just to get a feel for the stiffness of the vanes.

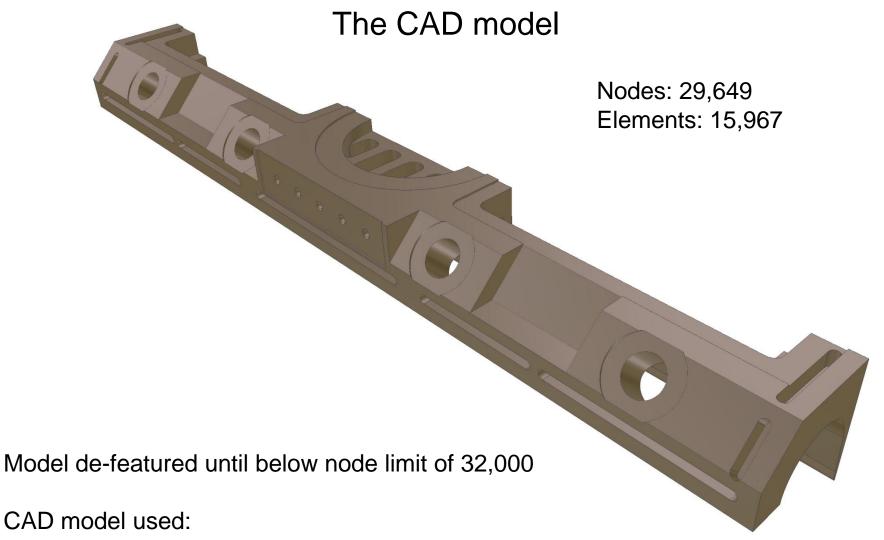












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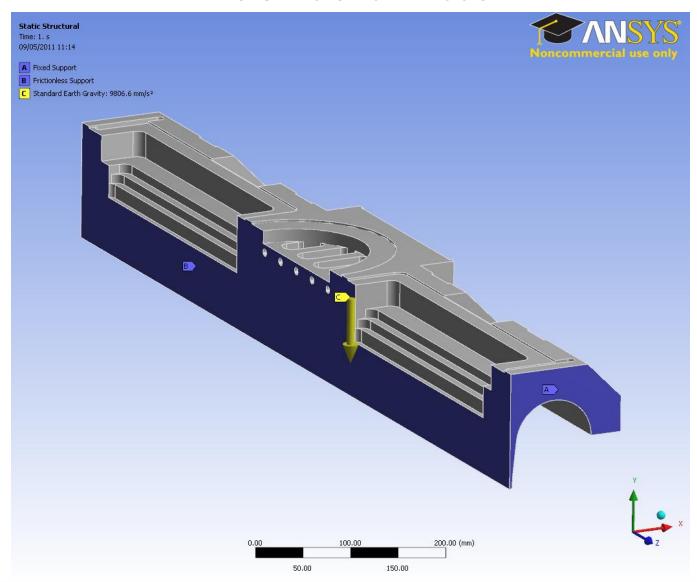








The simulation model





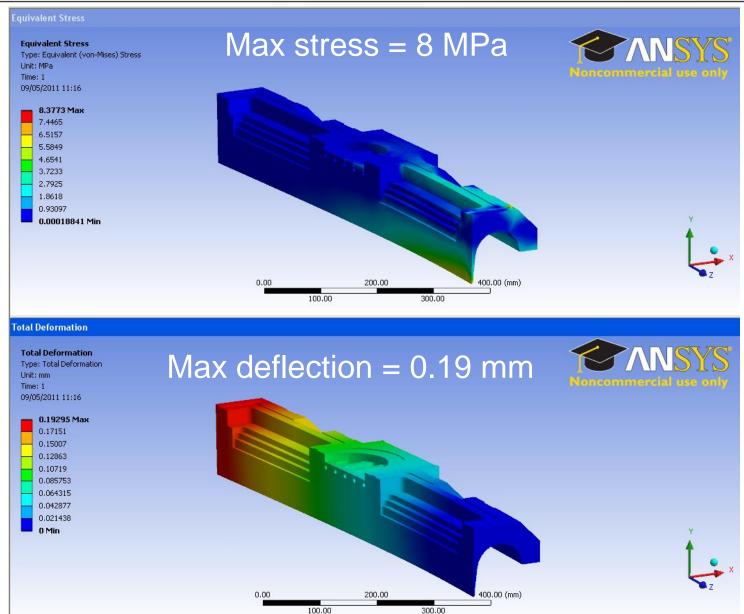
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Case 3

Lifting the complete 1m RFQ section in the cradle.





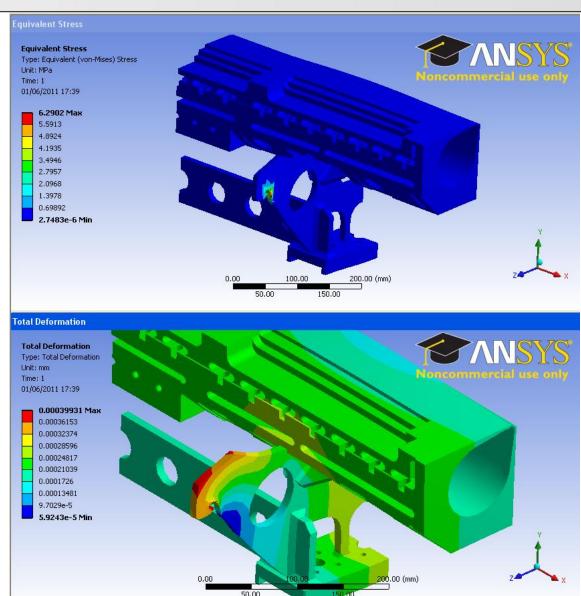






Max stress = 6.2 MPa

Max deformation = 0.0003mm













Use of tapped holes in Copper

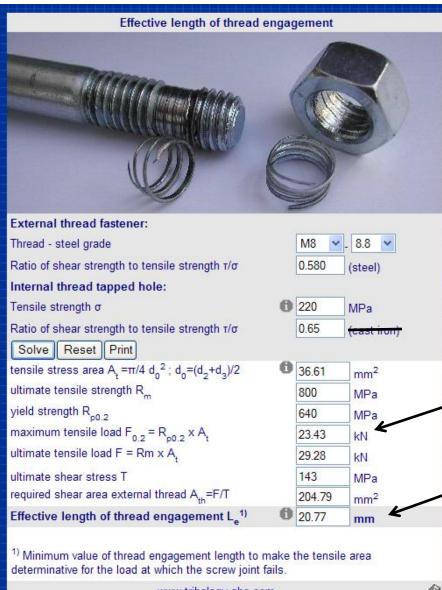


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Depth of tapped holes in Copper

Thread size	Thread depth		
M4	10 mm		
M5	13 mm		
M6	15 mm		
M8	21 mm		
M10	26 mm		

One M8 bolt can take 23kN (2.3 tonnes) tensile load.

Thread depth in Copper required to match the bolt tensile area = 21 mm.











Tapped holes in FETS RFQ – are they deep enough?

Tapped hole feature	Thread size	Recommend depth *	Actual depth
Tuner Port	M5	13 mm	14 mm +1 mm
Probe port	M4	10 mm	12 mm + 2 mm
Assembly blocks	M4	10 mm	10 mm + 0 mm
Cooling channel covers	M6	15 mm	18 mm + 3 mm
Vacuum port cooling manifold	M6	15 mm	18 mm + 3 mm
Vacuum pump mounting	M8	21 mm	16 mm – <mark>5 mm</mark>
Lifting holes	M10	26 mm	26 mm + 0 mm

^{*}The minimum value of thread engagement to make the tensile area determinative for the load at which the screw joint fails.









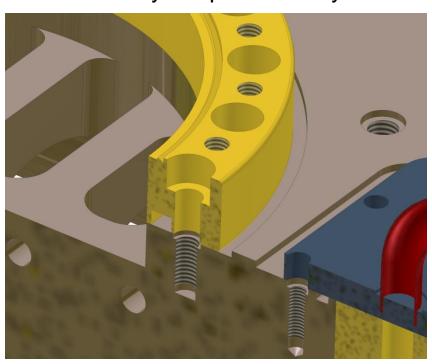


Tapped holes in FETS RFQ – are they deep enough?

One of our features has tapped holes that do not meet the minimum recommended depth.

Why?

Because any deeper and they would break into the cooling holes.



Should we be concerned?

- Each hole is 75% of the recommended minimum depth.
- There are 20 holes.
- •Only required to fully compress O ring.
- •Withstand pump vibration over time.
- Plus potential pump seizure.
- •Use of interim stainless steel flange which will be bolted on once and not removed.











General rules for RFQ assembly

- 1. By competent person only.
- 2. Clean parts and tools.
- 3. Check all threads are free running and bolts can be screwed in to full depth by hand before attempting to attach item.
- 4. Only small end of allen key required.
- Threads to be lubricated.
- 6. Torques to be specified.